

## Lecture 7

3<sup>rd</sup> Semester M Tech. Mechanical Systems Design

Mechanical Engineering Department

Subject: Advanced Engine Design

I/C Prof M Marouf Wani

Topic: Design Of Sports Car Engine

A Spark Ignition Engine Design

**Objective: Estimate Engine Displacement Volume Required – 01-10-2020**

**Numerical Example:**

**Q1 Design a Sports Car Engine.**

The engine is to have a rated power of 100 KW at 7200 rpm.

**Solution:**

The engine has to be designed for a sports car:

This is an S.I engine based Sports Car

Given Data:

**Rated Power = 100 KW**

**Rated Speed = 7200 rpm**

We Know - The best possible Brake Specific Fuel Consumption for S.I engine = 270 g/KWh

**Let**

**BSFC = 320 g/KWh**

Reasons –

Refer to the previous design of an engine for automobile application.

1. Rated Power = 100 KW
2. Rated Speed = 5500 rpm
3. BSFC = 300 g/KWh

When we compare the previous design numerical with this design numerical we can see the following:

Power: same for automobile engine and the sports car engine

Rated Speed: Higher for the Sports Car engine.

$$BSFC = \frac{\dot{m}_f}{P}$$

Higher rated speed for sports car means:

1. More number of power cycles per unit time.
2. More number of power cycles per unit time means mf will be higher

Since power for both the cases:

P = same

$\dot{m}_f$  = higher for sports car

**Let**

**BSFC = 320 g/KWh**

**Let**

**Volumetric Efficiency = 90 percent**

Reason:

1. The intake manifold design is better in terms of the volumetric efficiency, this is known as tuned intake manifold
2. The valve train design is better

**By using the equation for the definition of BSFC**

$$\text{BSFC} = \frac{\dot{m}_f}{P}$$

Where

$\dot{m}_f$  = mass flow rate of fuel

P = Power developed by the engine

Or

$$\frac{\dot{m}_f}{P} = 320 \text{ g/KWh}$$

**Power = 100 KW**

**(Decided as per engine application and comparison with previous example)**

Therefore substituting above:

$$\dot{m}_f = 320 \text{ g/KWh} * 100 \text{ KW}$$

$$\dot{m}_f = 32,000 \text{ g/h}$$

$$\dot{m}_f = 0.533 \text{ Kg/min}$$

**Mass flow rate of fuel = 0.533 Kg/min**

**The above computed data will help us to design the fuel supply system**

**Again**

**Let A/F ratio = 12.5**

Reasons:

1. The Operating Range of A/F ratio for S.I engines is (12 to 16)
2. The Stoichiometric A/F ratio for Petrol fuel = 14.6

3. The engine produces maximum power with slightly rich mixtures

**4. We are designing the engine for maximizing power**

**Therefore Let A/F = 12.5**

$$A/F = \frac{\dot{m}_a}{\dot{m}_f}$$

Where

$\dot{m}_a$  = mass flow rate of air

$\dot{m}_f$  = mass flow rate of fuel

From the above equation:

$$\dot{m}_a = A/F * \dot{m}_f$$

$$\dot{m}_a = 12.5 * 0.533$$

**Mass flow rate of Air = 6.662 Kg/min**

**The Above computed data will help us in the design of air supply system**

Now by using the equation for volumetric efficiency we can calculate engine displacement Volume required.

**Volumetric Efficiency is given by the equation:**

$$\eta_v = \frac{2 * \dot{m}_a}{\rho_{a,i} V_d N}$$

Where

$\dot{m}_a$  = Actual mass flow rate of air

$\dot{m}_a = 6.662$  Kg/min

$\rho_{a,i}$  = ambient inlet air density

$\rho_{a,i} = 1.18$  Kg/m<sup>3</sup>

$V_d$  = Engine displacement volume ---- ?

**$V_d = 1700$  cc**

**Displacement Volume Required = 1.7 liters**

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**In charge Course:**

**Prof M Marouf Wani**

Mechanical Engineering Department

National Institute of Technology

Srinagar, J&K

India – 190006

Text Book:

Vehicular Engine Design

By Kevin L. Hoag

Published By: SAE International USA